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1/77

Request for grant of a patent

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-5 SEP 2002

1.	Your reference	P31824/JED/JDB
2.	Patent Application Number *(the Patent Office will fill in this part)*	0220626.6
3.	Full name, address and postcode of the or of each applicant *(underline all surnames)*	Robert Gordon University Schoolhill Aberdeen AB10 1FR
	Patents ADP number *(if you know it)*	822 0816002
	If the applicant is a corporate body, give the country/state of its incorporation	United Kingdom
4.	Title of the invention	"Apparatus for Controlling the Launch, Secure Positioning and/or Recovery of Marine Based Equipment Situated in Sea or River Currents"
5.	Name of your agent *(if you have one)*	Murgitroyd & Company
	"Address for service" in the United Kingdom to which all correspondence should be sent *(including the postcode)*	165 - 169 Scotland Street Glasgow G5 8PL
	Patents ADP number *(if you know it)*	1198015

		Country	Priority application number *(if you know it)*	Date of filing *(day / month / year)*
6.	If you are declaring priority from one or more earlier patent applications, give the country and the date of filing of the or of each of these earlier applications and *(if you know it)* the or each application number			

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7.	If this application is divided or otherwise derived from an earlier UK application, give the number and the filing date of the earlier application		

8.	Is a statement of inventorship and of right to grant a patent required in support of this request? *(Answer 'Yes' if:* *a) any applicant named in part 3 is not an inventor, or* *b) there is an inventor who is not named as an applicant, or* *c) any named applicant is a corporate body.* *See note (d))*	Yes

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Priority documents	-
Translations of priority documents	-
Statement of inventorship and right to grant of a patent	-
Request for preliminary examination and search (Patents Form 9/77)	-
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11. I/We request the grant of a patent on the basis of this application

Signature MURGITROYD & COMPANY Date 04/09/2002

12. Name and daytime telephone number of person to contact in the United Kingdom — James D Brown 01224 706616

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"Apparatus for Controlling the Launch, Secure positioning and/or Recovery of Marine Based Equipment Situated in Sea or River Currents"

Technical field

The invention relates to a marine environment location device such as may be used for controlling the launch, positioning and recovery of a tidal turbine or other equipment. It should be noted that the example of a tidal turbine is used herein but the invention is not limited to such uses.

Background art

Tidal currents offer a considerable source of sustainable energy at various sites throughout the world, usually within easy reach of land and in relatively shallow waters. Tidal currents are created by movement of the tides around the earth producing a varying sea level, dependant on the phases of the moon and sun. As the sea levels vary,

so the waters attempt to maintain equilibrium subject to gravitational forces, thus inducing flow from one area of sea to another. This flow is modified by a number of factors such as, the Coriolis forces due to the earth rotation, earth/moon/sun alignment, local topography, atmospheric pressure and temperature and salinity gradients. The major advantage of tidal power generation is its regularity, which can be predicted for years in advance.

According to a study by the ETSU (Energy Technology Support Unit) the United Kingdom may obtain up to 20 percent of its total electricity by using these systems to collect energy from fast moving tidal currents that exist in channels and offshore areas. Similar resources have been noted to exist elsewhere such as in the Straits of Messina, between Sicily and mainland Italy.

The most powerful flows tend to occur in areas of restriction, either by width or depth, but for the same reasons are not suitable for widespread exploitation by large, fixed devices which require a minimum rotor area, and therefore water depth, to justify the costs of installation and maintenance. It is assumed from the outset that new tidal barrage systems are unlikely ever to be pursued due to their inherent properties of high cost, delayed financial return, and serious environmental consequences.

The considerable size of the available resource has attracted various proposals for its exploitation. The following represents the existing systems within the field of tidal current energy extraction. It is assumed that power transmission problems will be equal for any system, and that all systems will require some form of non-toxic anti-fouling agent.

There also exist operational environmental impacts common to all methods of tidal power generation, such as, an inherent risk of collision damage to fish and marine mammals, redirection of currents and the sediments and food particles contained within them, and shipping, particularly fishing.

A first type of tidal current energy extraction system encountered on the market is the Monopile system. This technology is well known and understood by contractors familiar with the offshore oil industry. It consists of twin axial flow turbines, each turbine driving a generator via a gearbox, mounted on streamlined cantilevers either side of a circular section, vertical steel monopile. It is anticipated that a number of structures will be grouped together in 'farms'. The planning of such a tidal 'farm' would need to be accurately modelled for wake effects, as once installed, the monopile is expensive to re-site. In addition, operational depth is restricted to the 20m - 35m range. Concerning the installation and maintenance, monopile systems require a hole to be drilled in suitable bedrock and the base of the turbine tower

is secured within the socket so produced. Existing monopile support mechanisms for presenting a tidal turbine to the tidal currents are expensive, thus making only a few sites economically viable for power generation and requiring considerable sub sea engineering expertise.

The current monopile systems permit raising the turbines above water level for maintenance and repair, which is beneficial, but the long-term (i.e. 20 years) reliability and corrosion resistance of the necessary mechanism must be questionable. The protrusion of the piles above sea level would reduce the likelihood of impact with passing vessels.

Concerning the environmental and decommissioning issues, the impact of installation would be considerable, especially to the benthic flora and fauna, but subsequently the piles may become areas of shelter and therefore, populated. To minimise the danger to shipping and fishing, decommissioning would require complete removal of the piles, which would disturb the benthic population once again.

A second type of tidal current energy extraction system that exists in the prior art is the floating tether. This floating tether device is anchored to the seabed with a mooring cable and suspended clear of the seabed using a flotation buoy. The axial flow tidal current turbine is free to position itself into the direction of the tidal flow, which obviates the need for a yaw mechanism.

Several prototypes have already been developed including a 10-kilowatt device tested in Scotland in 1994. At present, the arrangement is unlikely to be suitable for large power output installations due to the relative sizes of anchor, turbine and float. On occasions of relatively high velocity tidal streams (e.g. spring tides), if the anchor holds, the turbine will be dragged lower in the water with the unwanted potential to collide with the seabed.

Concerning the installation of the floating tether system, it is relatively quick and inexpensive. However, visual inspection would need to be frequent as the structure is likely to be subject to storm damage and fatigue loading of the cable, leading to possible loss of the supporting float and subsequent sinking of the device, or loss of anchorage and subsequent drifting. Once sunk, the device would be open to damage by the oscillating tidal currents and could prove difficult to recover, whilst a drifting device would potentially cause damage to any other moored turbines in its path.

Due to the length of tether required and the random positioning of the device at any one time, this arrangement is not suitable for closely grouped tidal farms and a safe spread would fail to make economical use of the power available in a given area. For the same reasons, this type of arrangement would present a hazard to all forms of shipping, large and small. It would, however

present a possible solution to a one-off, small scale installation in areas such as the mouth of a sea loch. Concerning the environmental impacts of installation and decommissioning of the floating tether systems, it will be minimal, leaving no footprint on removal.

A third type of tidal current energy extraction system that also exists in the prior art is the oscillating hydroplane system. In that system, a central post mounted on five legs supports a complex mechanism comprising two interconnected symmetrical hydrofoils. These hydrofoils are used to pump high-pressure oil, which drives an electrical generator via a hydraulic motor. At the end of each stroke, the hydrofoils are tilted to give the required angle of attack to produce the return stroke, thus creating an oscillating motion.

Concerning the installation and maintenance, at present, the oscillating hydroplane system does not yet possess a launch and recovery mechanism. As a result of the constant oscillations and considerable number of moving parts, it is probable that this device will be subject to high dynamic loading and subsequent fatigue stress. The upward stroke of the hydrofoils will tend to lift the device off the seabed and hence increase the possibility of it being washed away at high tidal stream velocities.

Concerning the environmental impacts of installation and decommissioning of the oscillating hydroplane

systems, they are expected to be minimal, leaving no footprint on removal. However, this cannot be confirmed until a launch/recovery mechanism is proposed. Using high pressure oil as a means of power transmission does however introduce the possibility of pollution in the event of leakage.

With these existing systems and designs, it is a problem that their instabilities during operations as well as during launch and recovery, if possible, might cause damage. In addition, since these systems are becoming larger and larger, the frequent installation and maintenance operations will become more and more difficult and expensive.

Summary of the invention

It is an object of the present invention to obviate or mitigate the problems of locating marine equipment in a flowstream by devising an apparatus capable of controlling the launch of said equipment, securing said equipment in position and permitting recovery for maintenance, repair or re-siting. The present invention also permits the launch and recovery to be carried out using a non-specialist but suitably equipped vessel.

In a first aspect, the invention described herein relates to an apparatus for controlling the launch and recovery of marine equipment attached to it, placed in sea or river currents for location and installation of said marine equipment, wherein said

apparatus comprises means for rotating through 180° for generating positive or negative lifts.

Said rotating means preferably comprises self-rectifying static hydrofoils, which may be capable of passive rotation about an axis such that it maintains alignment with a periodically reciprocating rectilinear flow. Typically, differences in pressure acting on its opposing surfaces due to a predetermined angle of attack causes said rotating means to generate negative or positive lifts.

Preferably, the rotating means is actuated by an actuating means such that it can be used to generate a variable positive or negative lift.

Preferably, said actuating means is a hydraulic, pneumatic or electric actuated motor.

Preferably, symmetrical hydrofoils can be used as rotating means.

Preferably, said apparatus for controlling the launch and recovery of marine equipment attached to it, is a support framework composed of sub frameworks on which are coupled said symmetrical hydrofoils.

Preferably, said apparatus for controlling the launch and recovery of marine equipment attached to it, is a multi-legged, self-levelling space frame

equipped with a number of hydrofoils, typically at different heights.

Preferably, the marine equipment attached to said space frame is positioned at or in as close proximity as possible to the centre of gravity of the apparatus.

Preferably, the space frame is mounted on a number of feet equipped with slippage prevention means, which may be an arrangement of spikes or the like, to typically resist slipping by shear force rather than relying on friction alone such that the negative lift will preferably tend to force said slippage prevention means into the seabed thus resisting the drag forces acting on the space frame tangentially to the seabed.

Brief description of the drawings

Embodiments of the present invention will now be described, by way of example only, with reference to the accompanying drawings, in which:-

Figure 1 shows a side view of a space frame in accordance with the present invention, showing a tubular frame allowing the positioning of the hydrofoils at differing heights;

Figure 2 in it's upper half shows the passive reversing of the hydrofoils in response to a change in flow direction whilst in it's lower half shows the different movements of

hydrofoils of figure 1 actuated by hydraulic motors to create positive and negative lifts during launch, recovery and transitional operations according to the present invention;

Figure 2a shows the passive reversing of the hydrofoils in response to a change in flow direction;

Figure 3 is a first side view showing the fundamental geometry of the passive reversing mechanism;

Figure 3a is a second side view showing the fundamental geometry of the passive reversing mechanism;

Figure 3b is a third side view showing the fundamental geometry of the passive reversing mechanism;

and

Figure 4 shows in details the assemblage of hydrofoils onto the space frame of Figure 1.

Detailed description of the invention

According to the present invention, the apparatus for launching a marine device from a vessel, securing the marine device whilst in operation on the seabed and permitting recovery to a vessel, for maintenance and repair should be as simple as possible without involving any sophisticated and specialised equipment. This is achieved by means of passive, self-rectifying static hydrofoils, the central shaft (see Figure 3) of which can be rotated

through 180° to generate positive or negative lift as required.

As is shown in Figure 1, the apparatus 1 for controlling the launch, secure positioning and recovery of a marine device comprises a space frame 10 for attaching to any desired marine device such as power extraction equipment which may comprise a tidal turbine (not shown), a hydrofoil support frame to accommodate the self rectifying hydrofoil mechanisms 12 and hydraulically operated legs 11 for levelling of the apparatus 1. The feet 14 are equipped with spikes or similar serrated attachments (not shown) to initiate grip on the sea or river bed.

The hydrofoils 12 are inclined in such a way as to generate a significant downforce as a result of the stream flow over their surfaces. This downforce will push the apparatus 1 into the seabed, and, since the actual applied force will be proportional to the square of the velocity of the fluid passing over them, the apparatus 1 will be more securely fixed as the streamflow velocity increases. By this means the apparatus can overcome overturning moments applied to the marine device that it supports.

The space frame 10 is shown as arched tubing but is not restricted to shape since any frame configuration offering different levels of mounting point for the hydrofoils 12 will suffice. The apparatus 1 as shown has multiple hydrofoils 12 but

any number of hydrofoils 12 will suffice. As is shown in Figure 2, each hydrofoil 12 is mounted on a central shaft 48 such that it may rotate upwards from horizontal (or any angle of inclination above horizontal) through vertical to any angle above horizontal but now pointing in the opposite direction. The angle of attack of the hydrofoils 12 is governed by the relative size and positioning of lugs 46 attached to the central shaft 48 and the corresponding lobes 44 attached to an outer shaft (not shown) which is itself fixed to the hydrofoil 12.

In a preferred embodiment, the apparatus 1 according to the present invention comprises a multi-legged, self-levelling space frame 10 equipped with a number of hydrofoils 12 at different heights with any marine device, such as a tidal turbine, it supports, situated as close as practicable to the centre of gravity of the apparatus 1.

It is anticipated that the space frame 10 will be mounted on a number of feet 14 equipped with spikes (not shown) to resist slipping of the apparatus 1 with respect to the river bed (not shown) by shear force rather than relying on friction alone. The number of feet 14A, 14B required will depend on the weight of the apparatus 1; however, the location and the shape of these supporting feet 14A, 14B aim at holding the apparatus 1 in the orientation shown in Figure 1 upwards against the current and thus ensuring the stability of the space frame 10. The

negative lift (arrow A) will tend to force these spikes into the sea or river bed (not shown in Figure 1) thus resisting the drag forces acting on the space frame 10 tangentially to the sea or river bed.

The drag forces acting on the marine device (such as the tidal turbine) attached to the apparatus 1 will naturally tend to apply an overturning moment to the space frame 10 about its rearmost feet 14B, with respect to the direction of flow (arrow F). These forces will however be overcome by positioning the hydrofoils 12 at stations such that the negative lift (arrow A), created by the foremost or upstream (those at the left hand side of the space frame 10 as shown in Figure 1) hydrofoils 12 acting over the length of the space frame 10, is arranged to exceed the overturning moment.

Thus, the space frame 10 is symmetrical about its midpoint M with the hydrofoils 12 being coupled to the space frame 10 in a manner, to be subsequently detailed in a discussion of Figure 2, which allows them to passively reverse with stream flow F to maintain compressive forces in a downwards direction A and restraining moments regardless of tidal stream direction.

During operation of the apparatus 1, the hydrofoils 12 are free to rotate (shown as clockwise in the upper half of Figure 2 and 2a) in response to the change in tidal stream flow F direction in a manner

which is shown from left to right in the upper half of Figure 2 to create a negative lift (arrow A) so as to push the apparatus 1 into the seabed.

When the apparatus 1 is to be installed on the seabed or is to be recovered from the seabed for e.g. maintenance of the apparatus 1, as shown in the upper part of Figure 2, hydraulic motors 30, via a suitable gearing mechanism such as a worm and wheel arrangement 32 (as shown in Figure 3) or chain type mechanism (not shown), are utilised to rotate (shown as anticlockwise in the lower half of Figure 2) the longitudinal axes (i.e. the horizontal axes perpendicular to the stream flow 12) of the hydrofoils 12 through the required angle until the hydrofoils 12 have reached the configuration shown at the right hand side of the lower half of Figure 2; for the configuration shown in the lower half of Figure 2, this angle is approximately 180°. It should be kept in mind that the hydraulic motors 30 can be replaced by pneumatic or electric motors. In other words, if the apparatus 1 is towed, e.g. by a boat or other vessel or installation at the surface, the hydrofoils 12 will produce positive lift (arrow B) as shown in the lower half of Figure 2. For launch and recovery, this positive lift can be utilised to raise or lower the space frame 10 within the tidal stream. If required, this action could be augmented by forming air tanks within the space frame 10 that can be 'blown' with compressed air to improve the buoyancy of the apparatus 1. If the hydraulic motors 30 use the worm and wheel mechanism

32 form of drive, the hydrofoil 12 positions can be altered over a range of positions, thus permitting the apparatus 1 to be 'flown' in the water. Hydraulic connections (and pneumatic connections if required) can be affixed to a supporting marker buoy (not shown) for ease of access.

Figure 3 shows the mechanism and assemblage of hydrofoils 12, hydraulic motors 30 and worm and wheel drive shaft mechanisms 32 in more detail. The hydrofoils 12 are free to rotate about a central shaft 48, through an included angle of say 160° which will maintain an angle of 10° to the horizontal. The 10° angle effectively becomes an angle of attack when the tidal stream flow F reverses. Thus as the tidal stream 10 reciprocates, the hydrofoils 12 will maintain an angle of 10°, creating a negative lift (arrow A), which will therefore push the spikes 16 into the seabed and immobilise the space frame 10. As will be described subsequently, positioning lugs 46 mounted on a central shaft 48 provided a stop for locating lobes 44 of the hydrofoil 12, such that the hydrofoil 12 cannot rotate further than the 160° shown in the upper half of Figure 2.

By rotating the central shaft 48 through slightly greater than 180° (say 200°), the negative lift becomes positive lift (arrow B) and the space frame 10 will rise through the water so that the tidal turbine 90 can be recovered on the vessel (not shown).

Figure 4 shows in more detail the mechanical assemblage of hydrofoils 12 with space frame 10. The hydraulic motor 30 for actuating the positioning gear is equipped with a drive shaft 32 that is utilised for rotating an indented positioning gear 42 or a toothed gear wheel. The positioning gear 42 is solidly attached to a central shaft 48 which passes through a bore provided in the larger end of each hydrofoil 12, a section of which is show on Figure 4. The bore of the hydrofoil 12 is provided with a pair of diametrically opposed and inwardly projecting hydrofoil locating lobes 44. The central shaft 48 has a pair of diametrically opposed and outwardly projecting positioning lugs 46, each one of which selectively co-operates with one of the respective pair of diametrically opposed hydrofoil locating lobes 44.

Thus, by rotating the drive shaft 32, the hydraulic motor 30 actuates or rotates the position gear 42 which in turn rotates the central shaft 48. The positioning lugs 46 will contact the locating lobes 44 and carry them 44 (and the hydrofoil 12) about the rotational axis of the central shaft 48 until the hydrofoil 12 is in the desired configuration, this being through an angle of approximately 160° until the hydrofoil 12 is in the configuration shown at the right hand side of the lower half of Figure 2. At this point, the motor 30 is de-actuated and the positioning lugs 46 will hold the hydrofoil 12 locked in this configuration. The rotation of 160°

enables the hydrofoil 12 to maintain an angle of 10° to the horizontal in order to provide an angle of attack when the tidal stream F reverses.

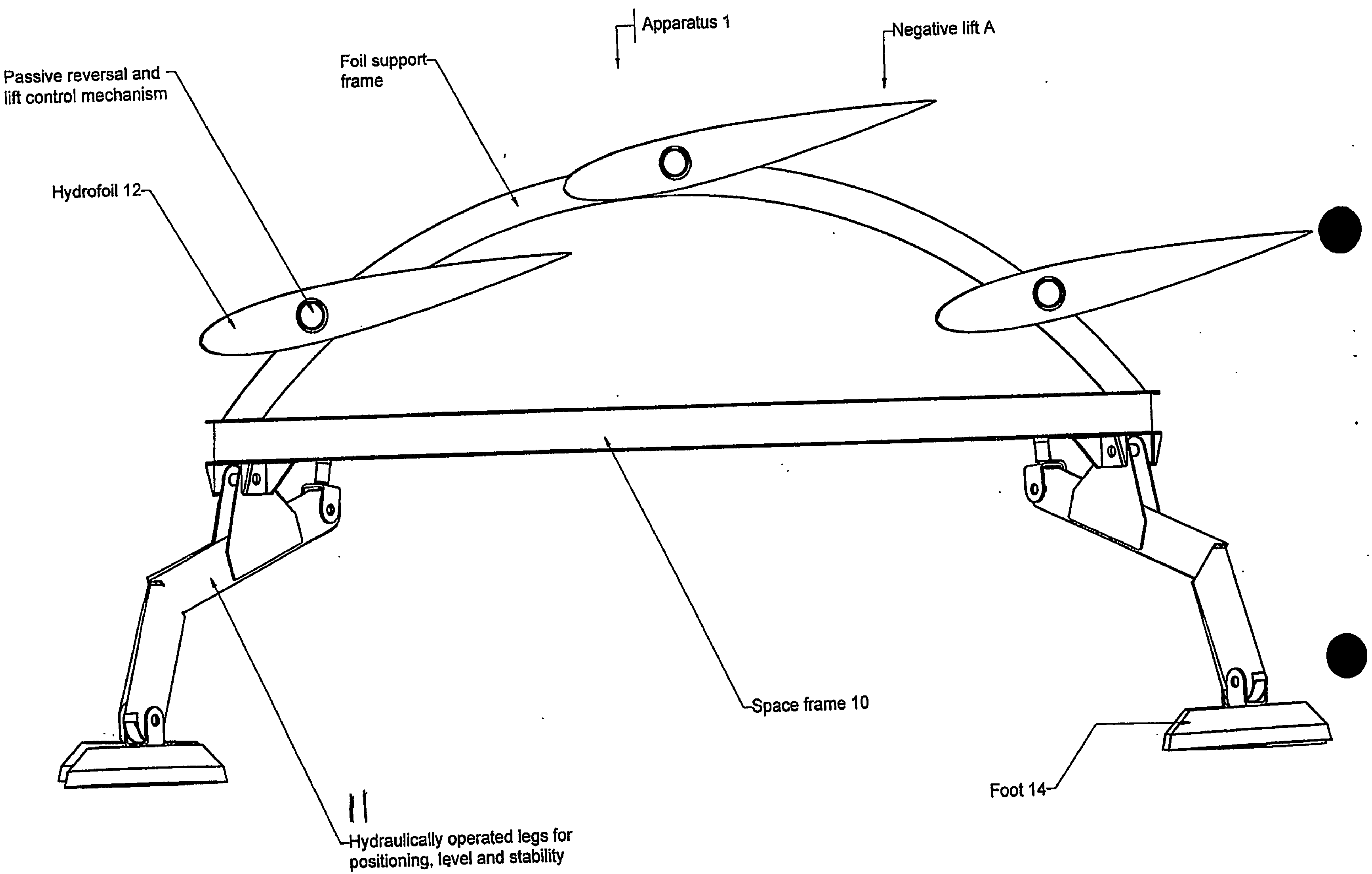
Conversely, the rotation of the central shaft 48 by 180° drives the hydrofoils 12 to create a positive lift and in which case, the space frame 10 will rise through water. Figure 3a shows how the attitude of the hydrofoil 12 is changed by a simple 180° clockwise rotation of the central shaft 48.

The apparatus according to the present invention, can be launched and recovered by a non-specialist vessel, using non-specialist equipment. Indeed if the vessel is large enough, a number of apparatus 1 may be launched or recovered in a day without the need to return to port. This will also permit easy access for maintenance and repair. Since apparatus 1 possesses few moving parts and no complex mechanisms, it should be inherently reliable.

Concerning the primary environmental impact of embodiments of apparatus 1 according to the present invention, it would have some impact upon the benthic flora and fauna, and, although the positioning and retrieval of apparatus 1 would be relatively frequent (at least once every year is anticipated), nothing more than temporary localised disturbance is anticipated. There exists some potential for hydraulic oil leakage, but the system contents are minimal so, even in the event of complete system evacuation, any oil contamination

would be minor. Operational environmental hazards are in common with the other forms of tidal energy extraction and decommissioning would leave no footprint.

Improvements and modifications in terms of dimensions and locations of the different parts described above may be incorporated to the hereinbefore described apparatus for controlling the launch and recovery of a tidal turbine without departing from the scope of the present invention.



Apparatus 1
Negative lift A
Passive reversal and
lift control mechanism
Foil support
frame
Hydrofoil 12
Space frame 10
Foot 14
Hydraulically operated legs for
positioning, level and stability

Figure 1

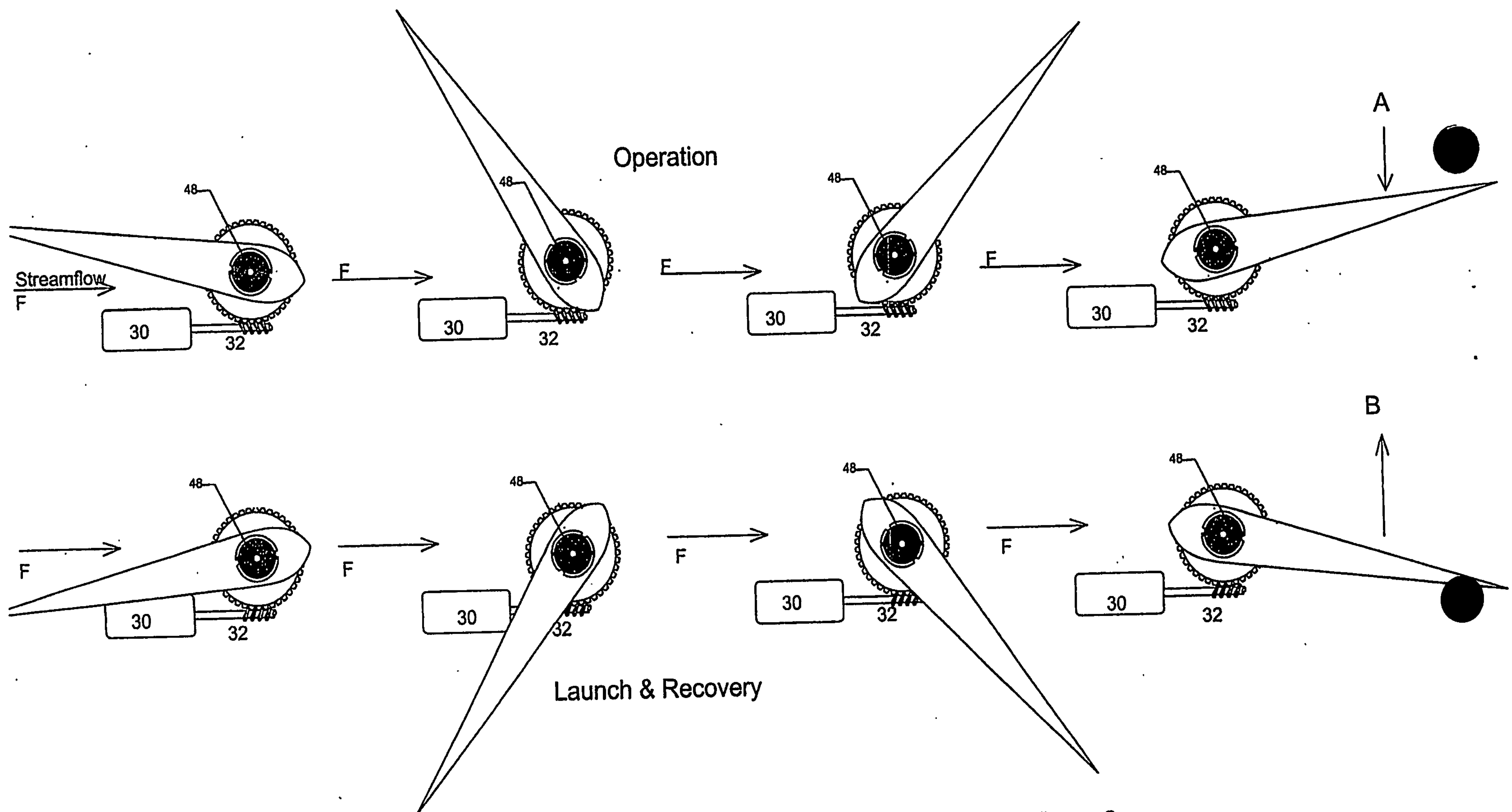


Figure 2

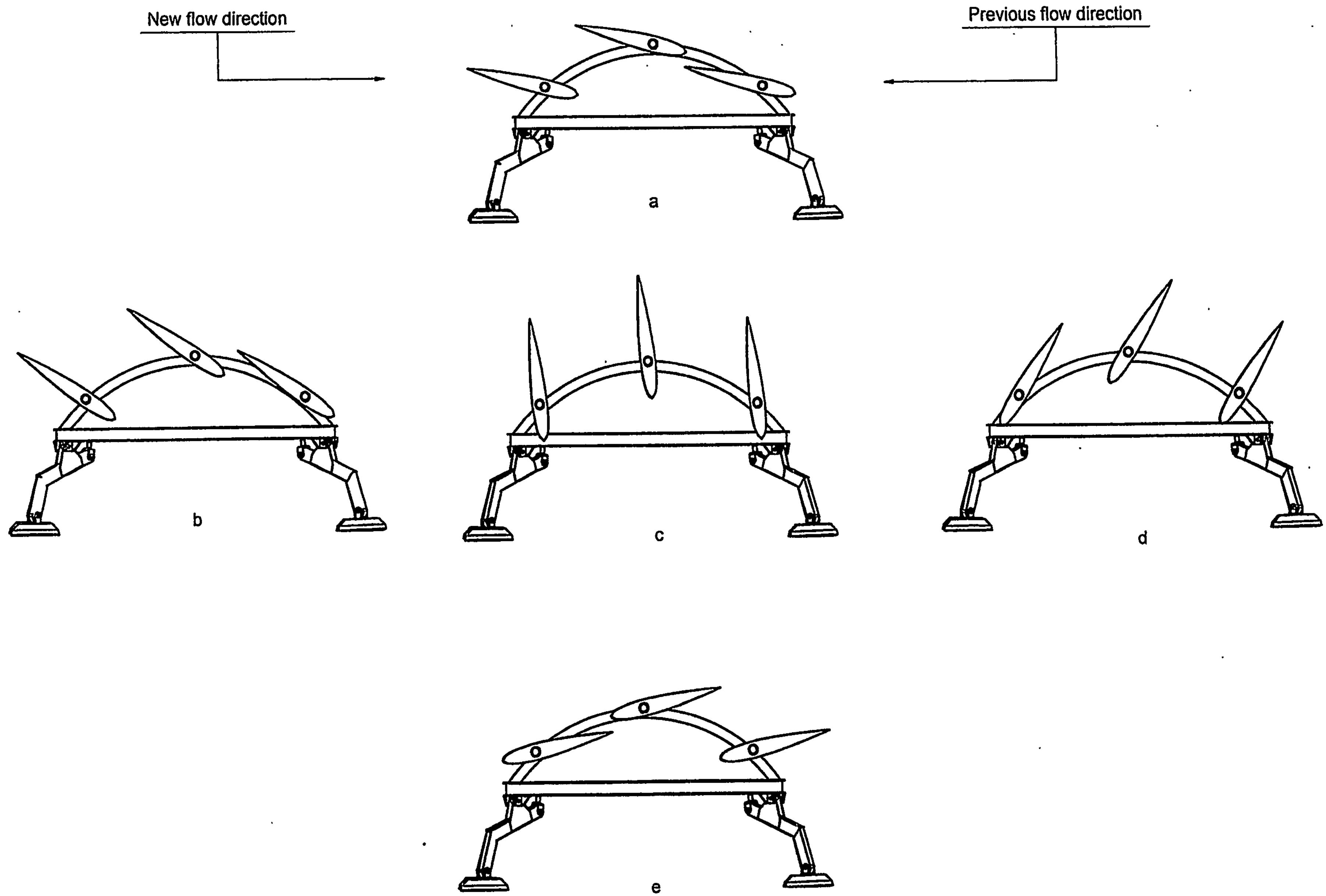


Figure 2a

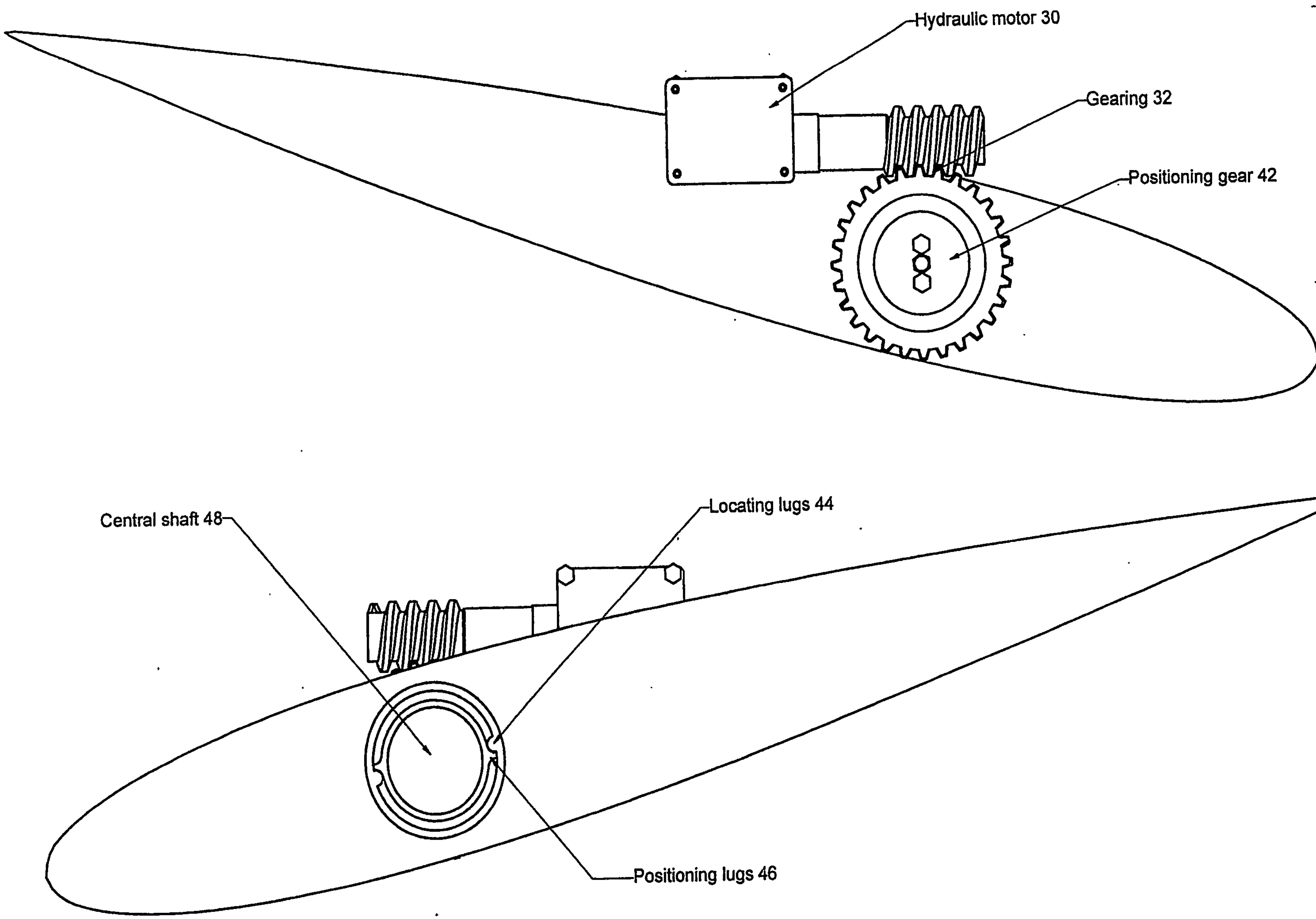

Hydraulic motor 30
Gearing 32
Positioning gear 42
Central shaft 48
Locating lugs 44
Positioning lugs 46

Figure 3

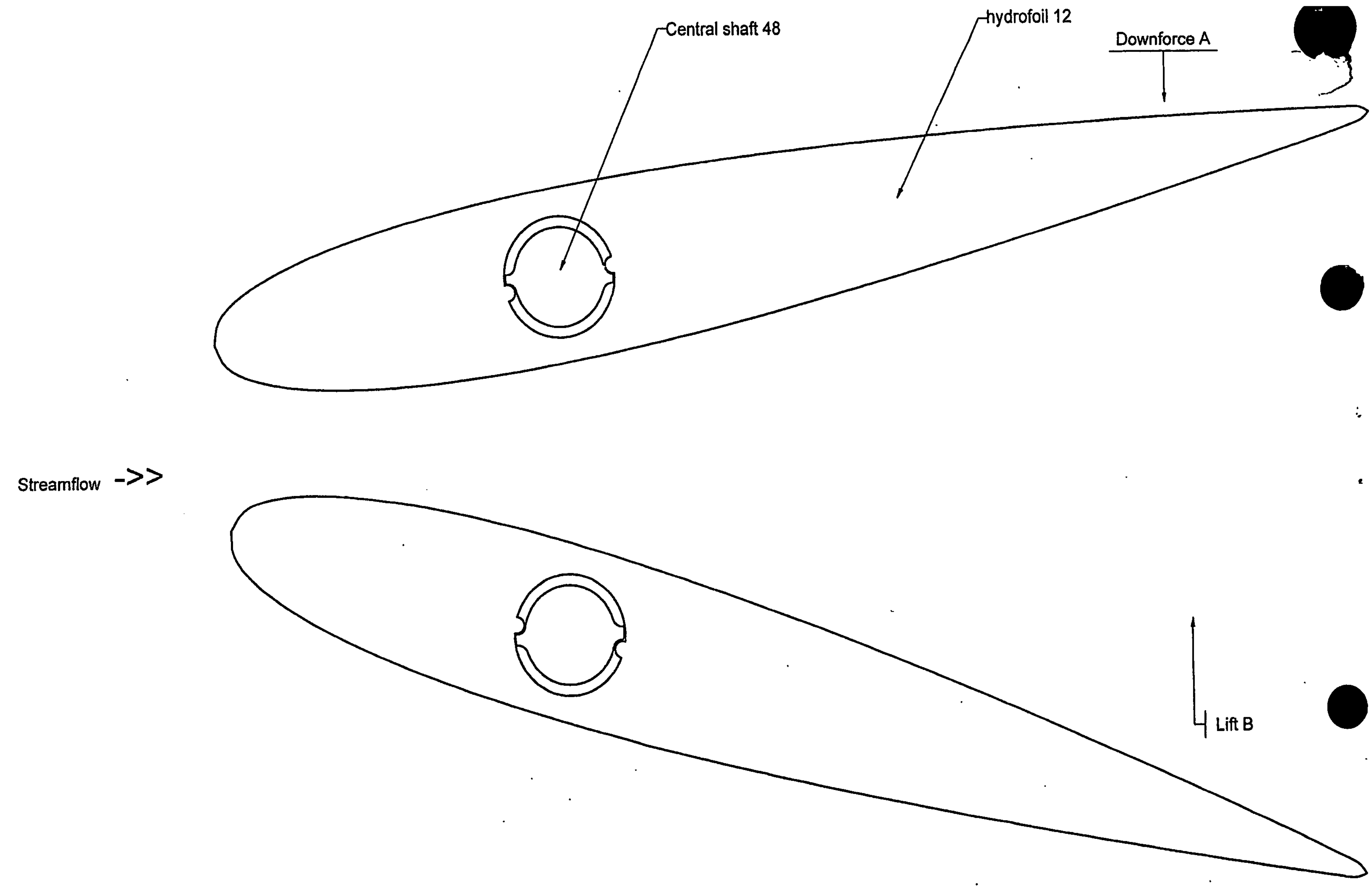
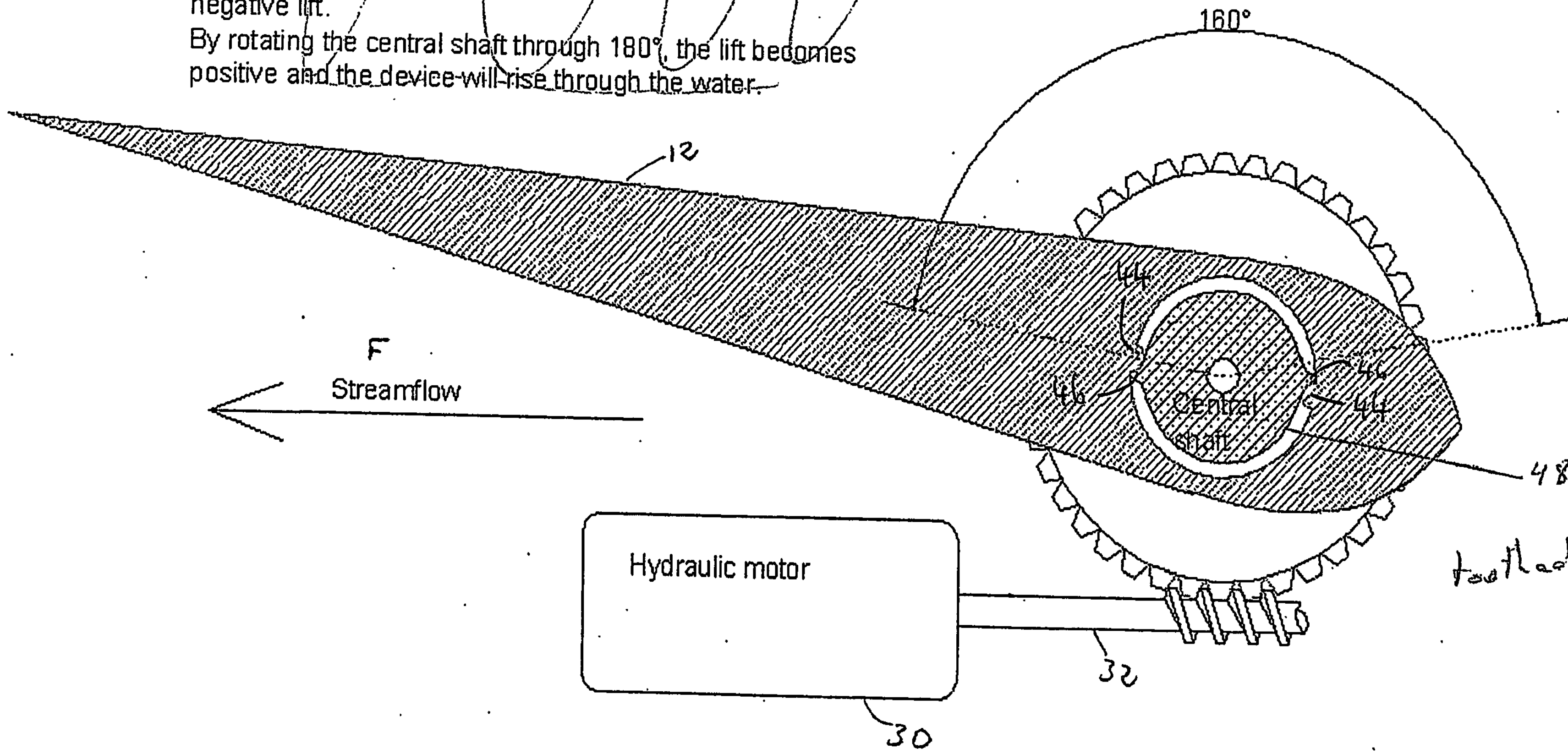

Central shaft 48
hydrofoil 12
Downforce A
Streamflow ->>
Lift B

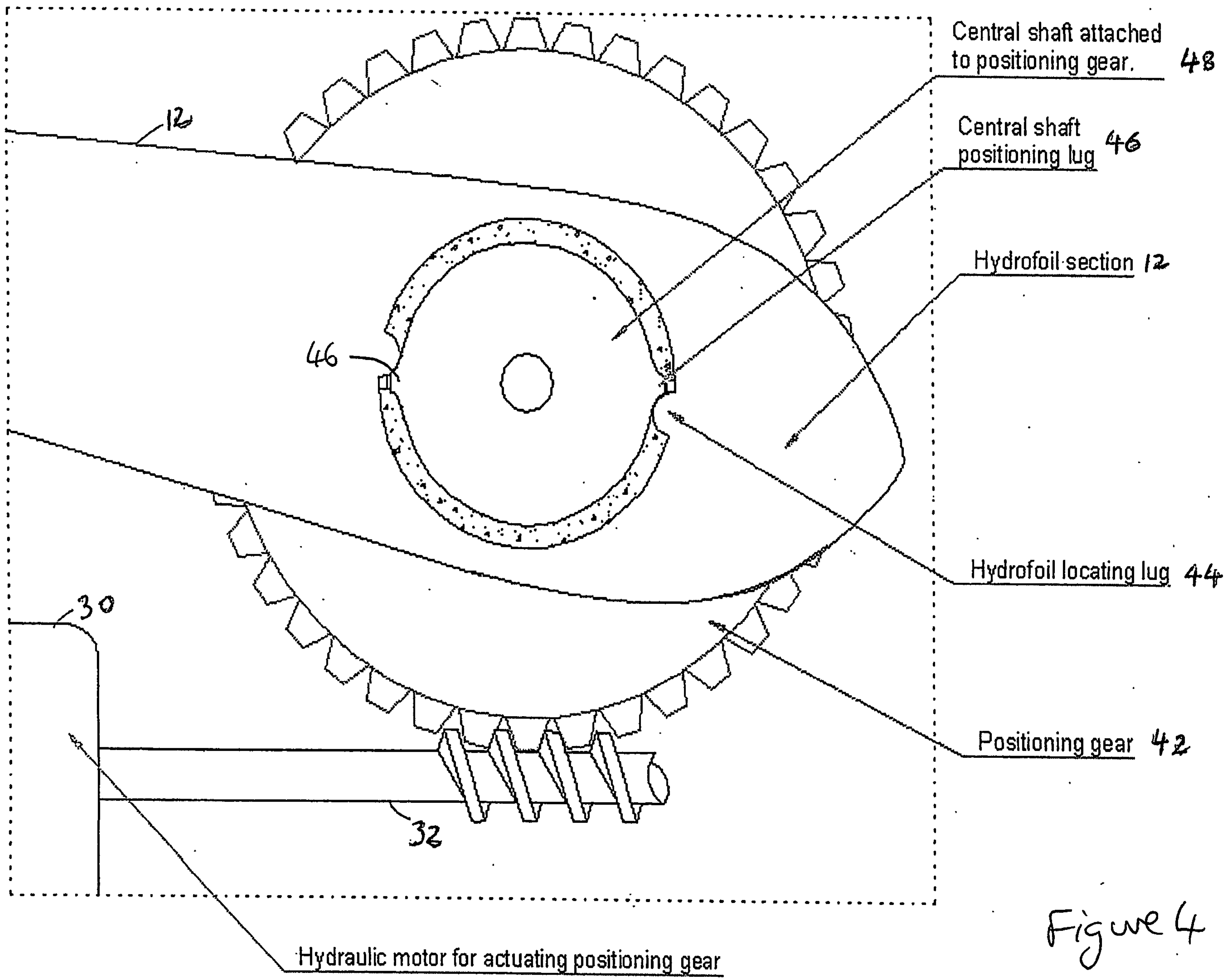
Figure 3a

Figure 3b

The foil is free to rotate about a central shaft, through an icluded angle of, say 160° which will maintain an angle of 10° to the horizontal. The 10° angle effectively becomes an angle of attack when the tidal stream reverses. Thus as the tidal stream reciprocates, the foil will maintain an angle of 10°, creating negative lift.

By rotating the central shaft through 180°, the lift becomes positive and the device will rise through the water.





Central shaft attached to positioning gear. 48
Central shaft positioning lug 46
Hydrofoil section 12
Hydrofoil locating lug 44
Positioning gear 42
Hydraulic motor for actuating positioning gear
12
46
30
32

Figure 4